



# WIND TURBINE AERODYNAMIC STUDY WITH OPENFOAM

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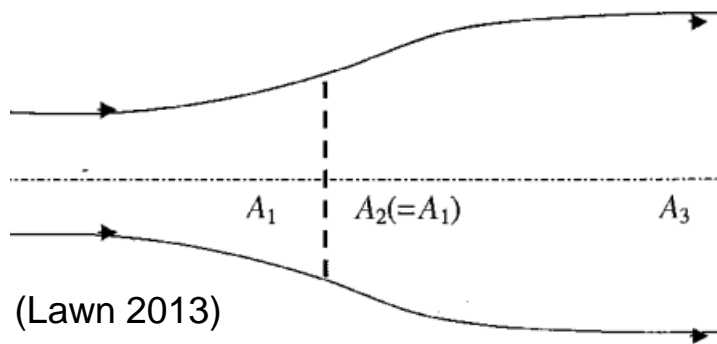
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# 1. Introduction

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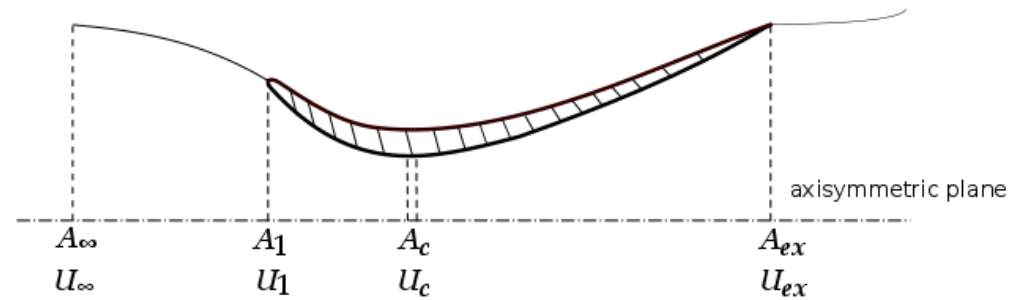
- The extractable energy of a horizontal-axis turbine rotor of fixed size can be increased by installing it at the entrance of a diffuser.
- The aim of this work is to analyze the influence of the pressure losses of a diffuser augmented wind turbine (DAWT) in the extractable power.

- The diffuser produces a greatly reduced pressure behind the turbine compared to the one behind a bare turbine.



(Lawn 2013)

*Bare turbine*

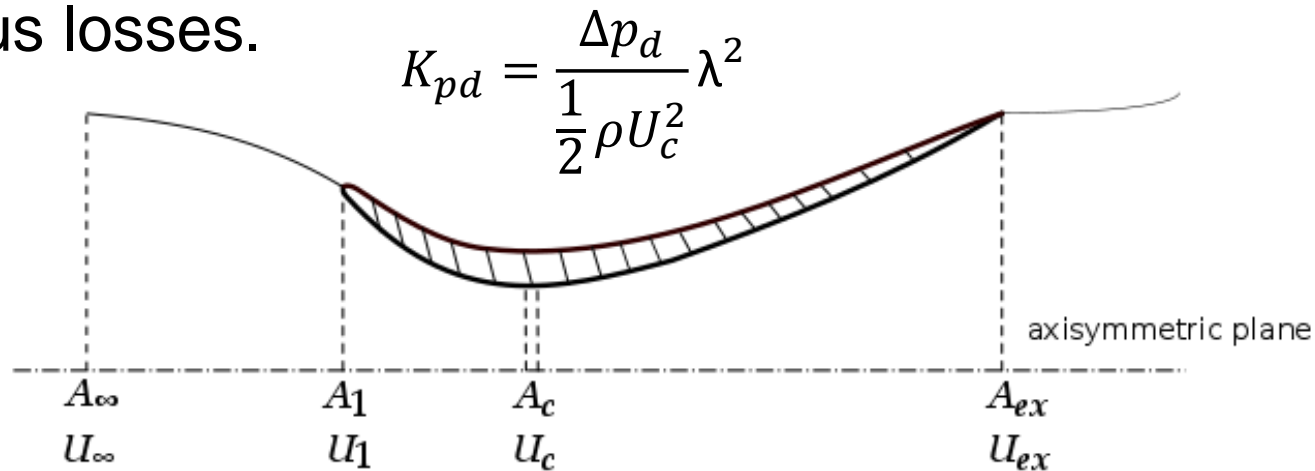


*Ducted Turbine*

- This effect increases the mass flow-rate through the DAWT duct, with at least as much pressure change as across a conventional turbine.

- The increase of mass flow-rate in the diffuser is influenced by four main factors:

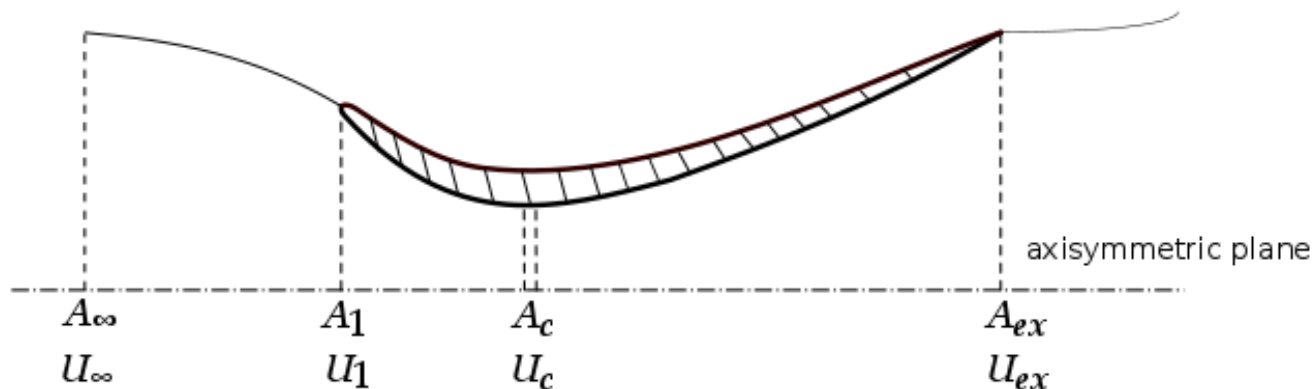
- The diffuser area ratio.  $\lambda = \frac{U_c}{U_{ex}} = \frac{A_{ex}}{A_c}$
- The base pressure reduction at the diffuser exit caused by the obstruction flow.
- The flow separation downstream in the diffuser.  $c_{pex} = \frac{p_{ex} - p_{\infty}}{\frac{1}{2}\rho U_{\infty}^2}$
- Viscous losses.



# 2. Analytical Framework

$$p_\infty + \frac{1}{2}\rho U_\infty^2 = p_{ex} + \frac{1}{2}\rho U_{ex}^2 + \Delta p_c + \Delta p_d,$$

$$K_{pc} = \frac{\Delta p_c}{\frac{1}{2}\rho U_c^2} \lambda^2, \quad c_{wc} = \frac{\Delta p_c U_c A_c}{\frac{1}{2}\rho A_c U_\infty^3} = K_{pc} \lambda \left( \frac{U_{ex}}{U_\infty} \right)^2.$$



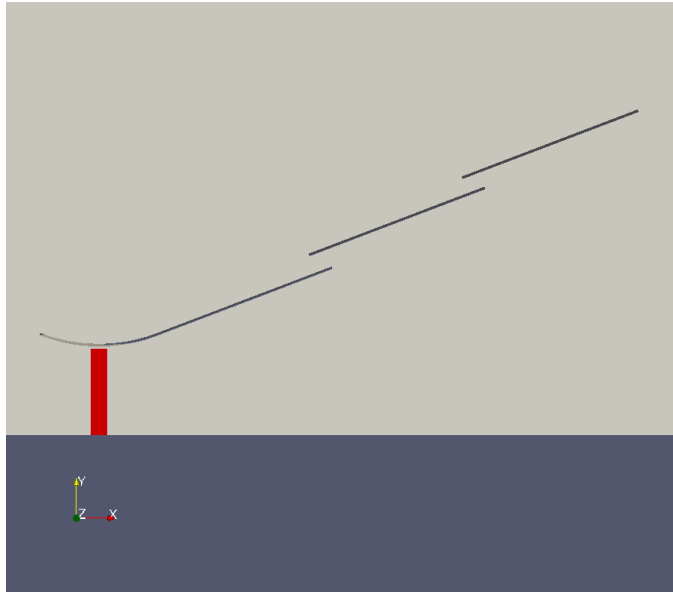
- Parameter to compare different DAWT duct configurations

$$c_{Wc} = \frac{\Delta p_c U_c A_c}{\frac{1}{2} \rho A_c U_\infty^3} = K_{pc} \lambda \left( \frac{U_{ex}}{U_\infty} \right)^2 .$$

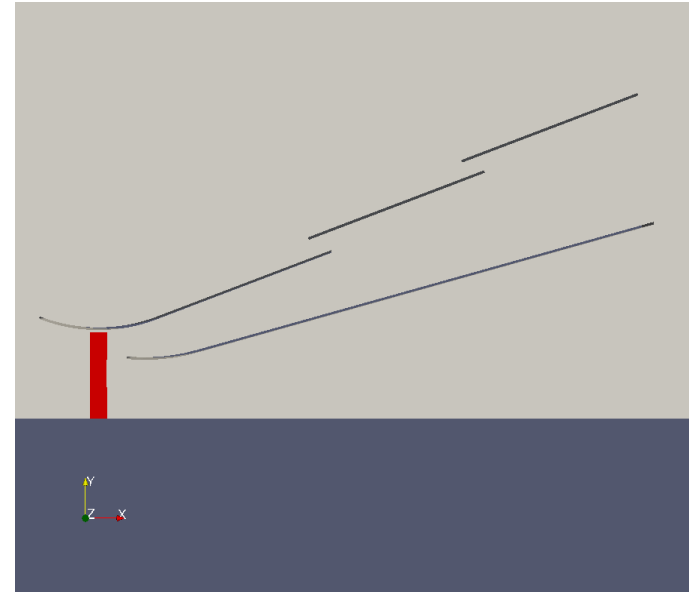
- Betz's Limit:  $c_{Wc} = 0.593$  ,  $K_{pc} = 2$
- Objective  $\longrightarrow$  Maximize  $c_{Wc}$

# Analyzed Configurations

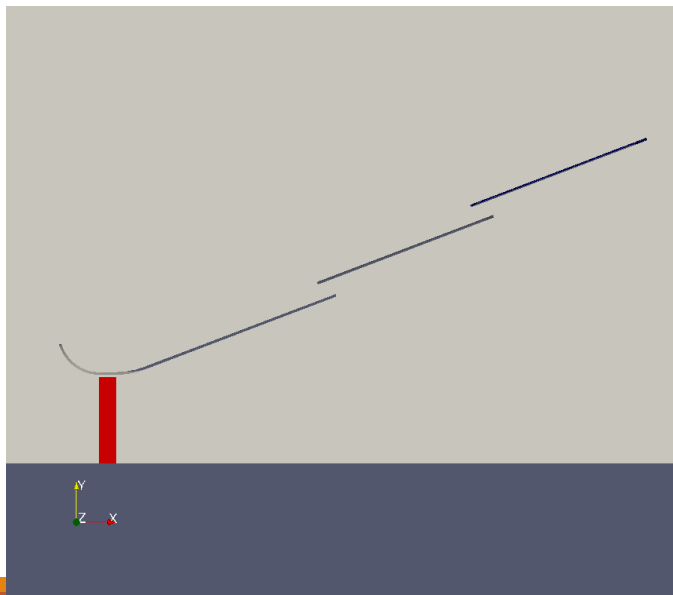
1a



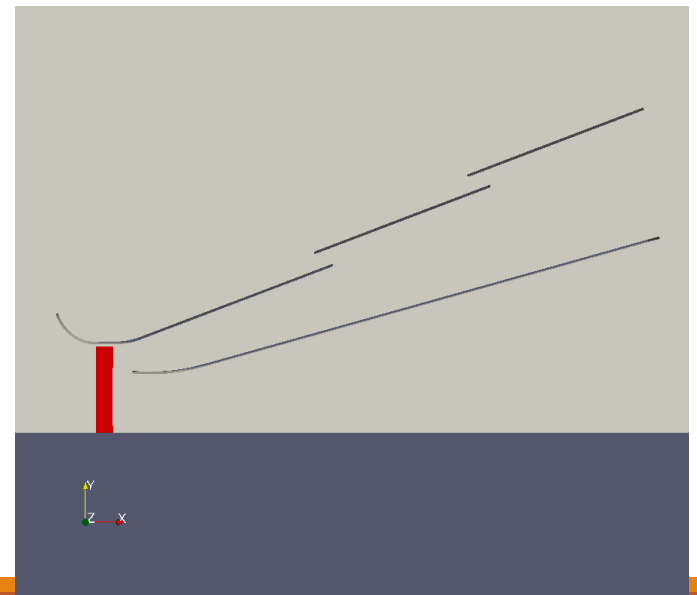
1b



2a



2b



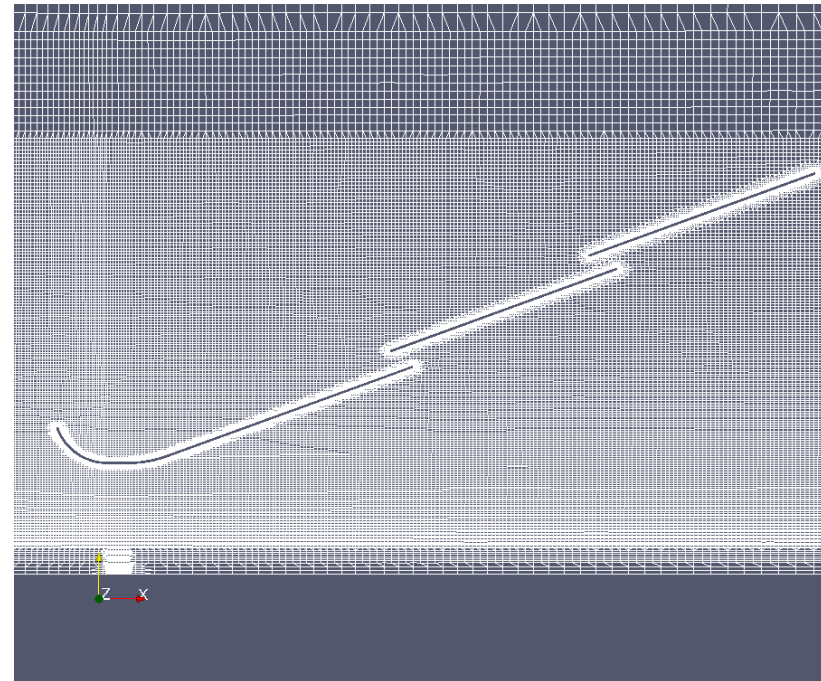
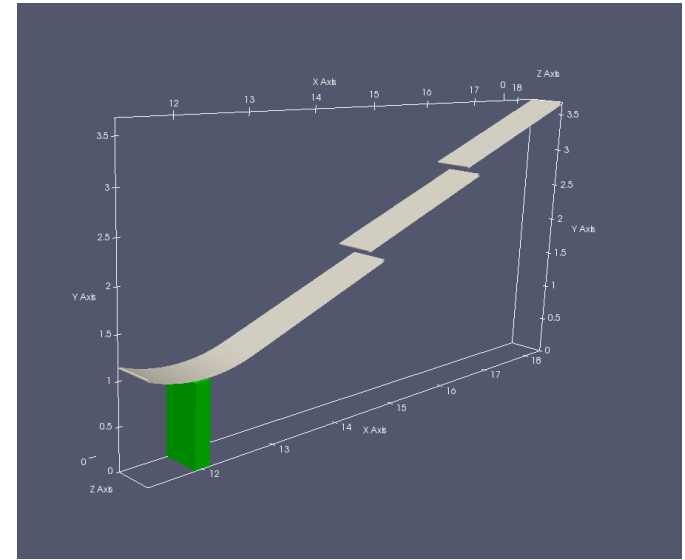
# 3. Numerical Setup

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- Steady RANS equations
- Solver: porousSimpleFoam
- Scheme: 1<sup>st</sup>
- Turbulence Model:  $k - \varepsilon$
- Reynolds number:  $Re \approx 550\,000$  on the turbine section
- Freestream velocity:  $U = 4$  m/s

# Computational Domain

- 60x24x0.5 m
- Geometry generation with Salome and python
- Mesh generation with snappyHexMesh
- N° cells  $\approx$  130 000
- Duct length 17 m
- $\lambda = \frac{U_c}{U_{ex}} = \frac{A_{ex}}{A_c} = 3.7$

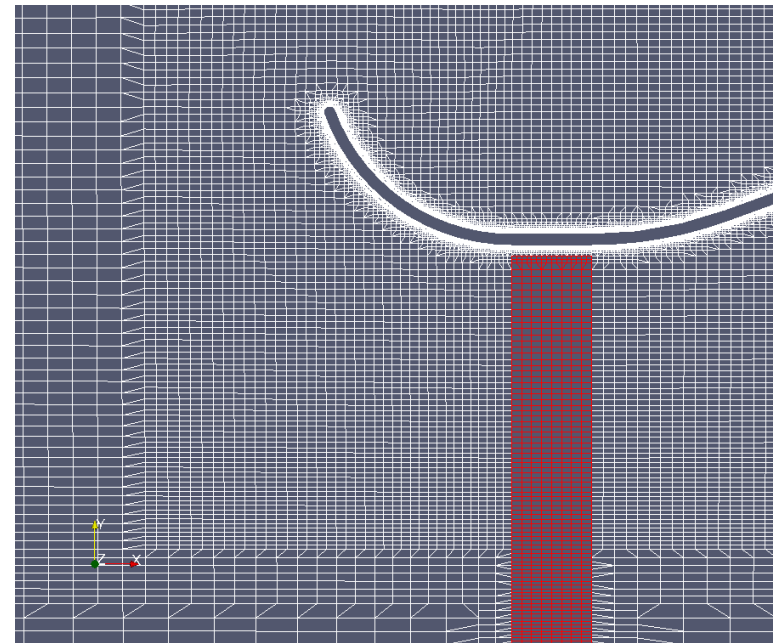


# Porous Region

- 2D model, using a porous region as an actuator disk
- Darcy-Forchheimer formulation
- Just inertial terms ( $F$ ) were considered
- Momentum equation in this region:

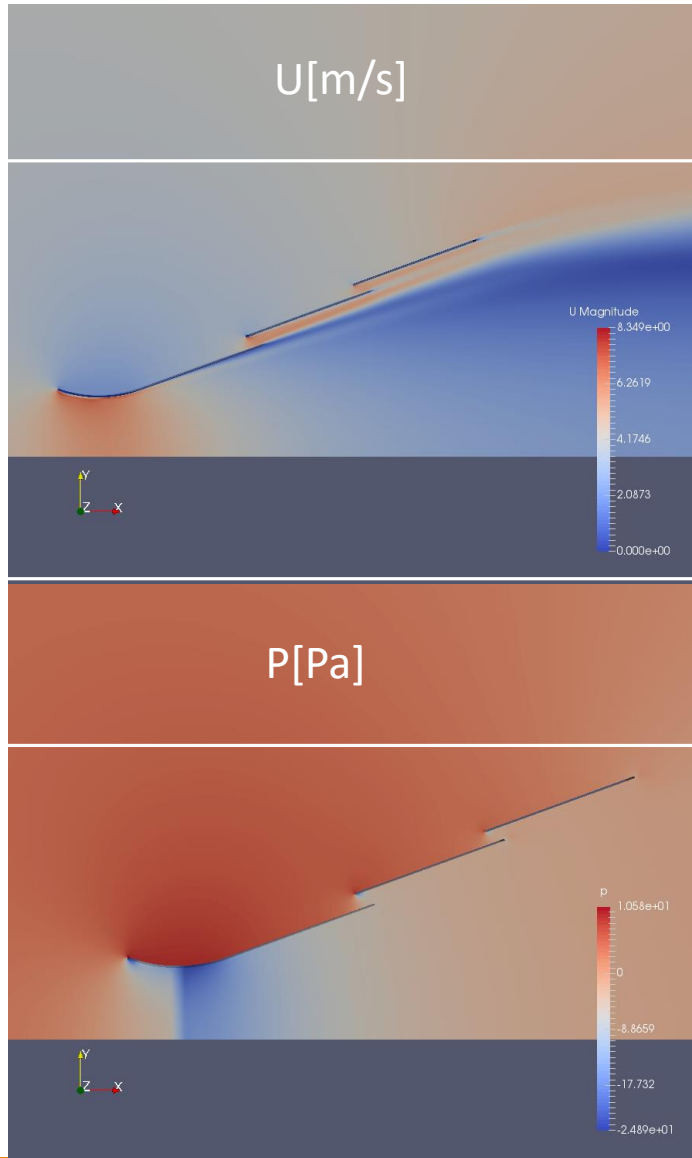
$$u_j \frac{\partial}{\partial x_j} (\rho u_i) = -\frac{\partial p}{\partial x_i} + \mu \frac{\partial \tau_{ij}}{\partial x_j} + S_i$$

$$S_i = -\left( \mu D + \frac{1}{2} \rho |u_{jj}| F \right) u_i$$

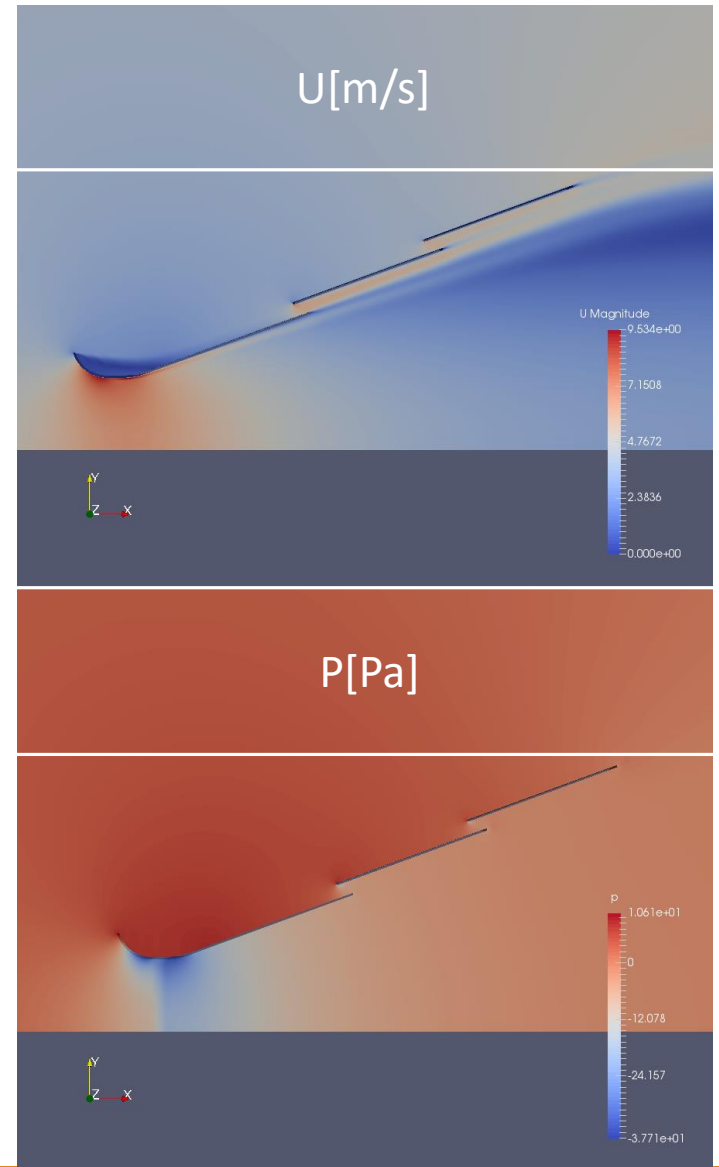


# 4. Results

1a

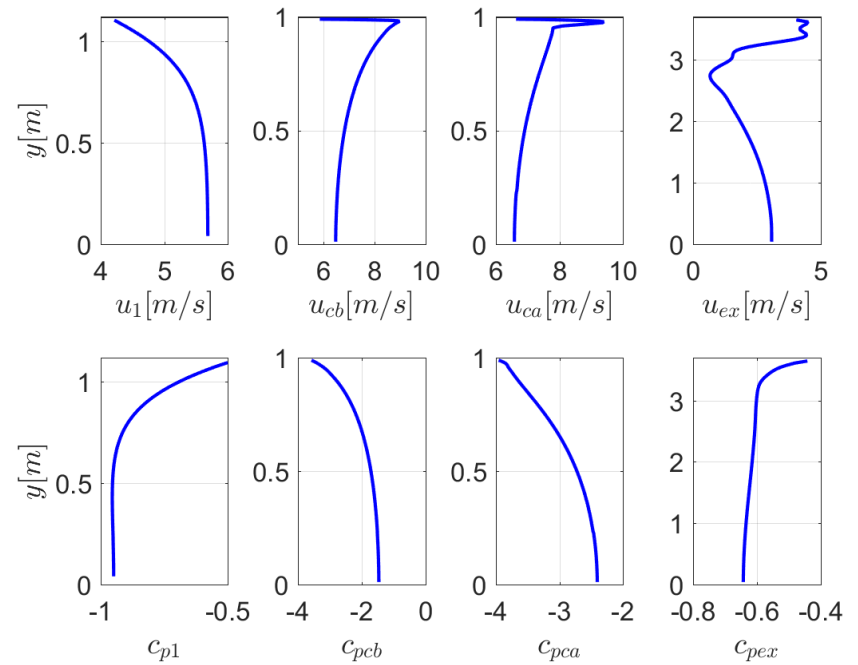
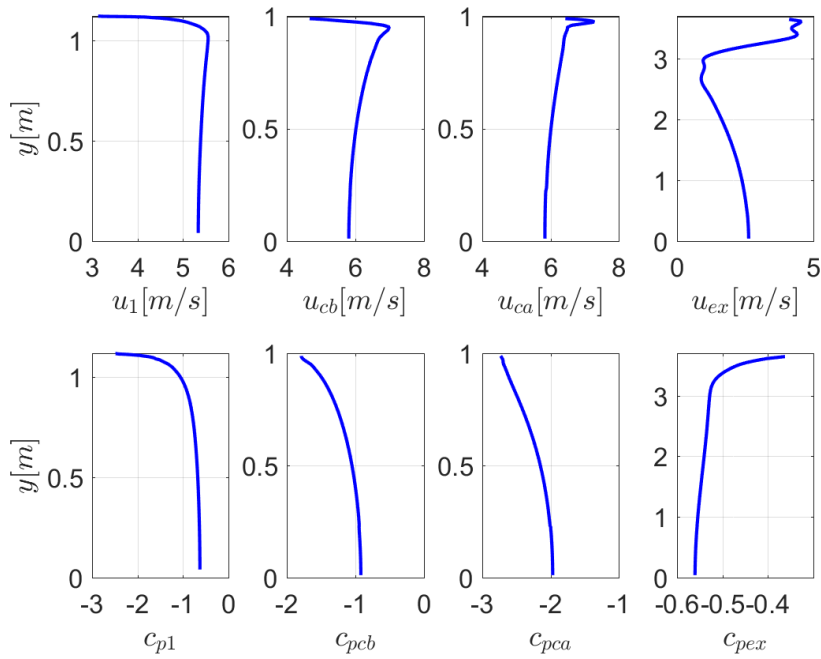
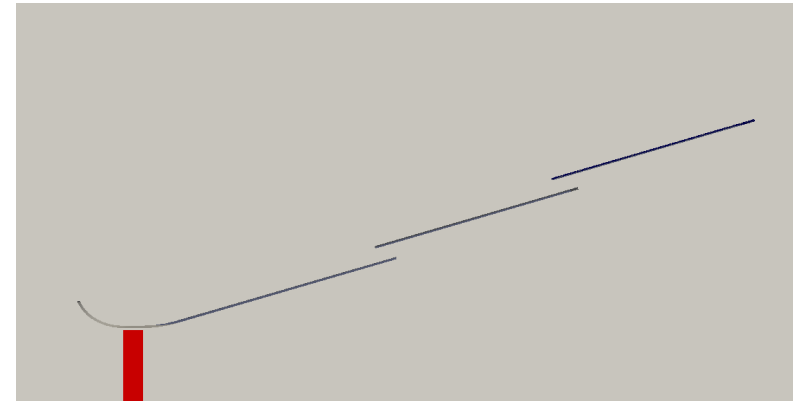
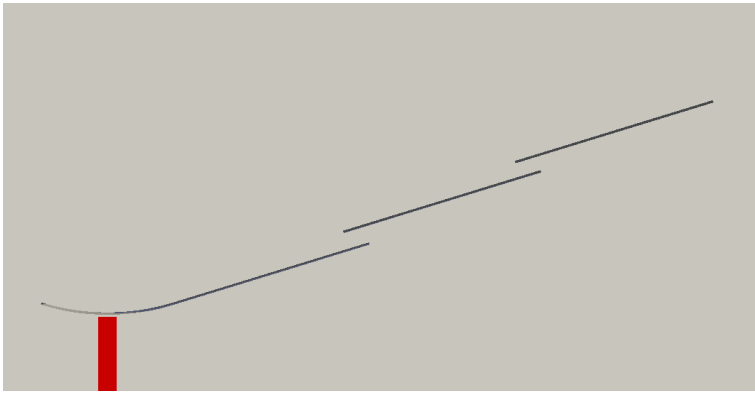


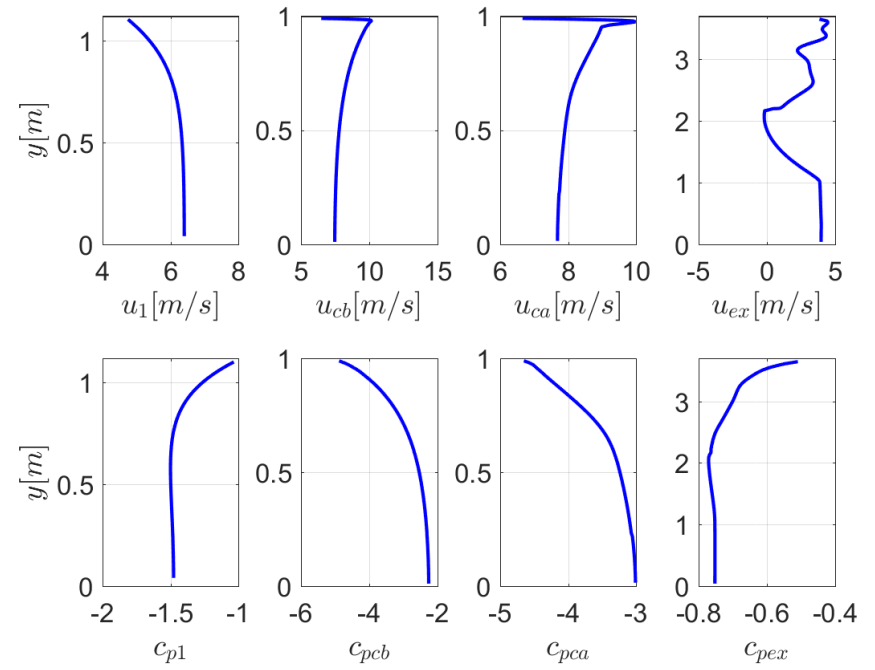
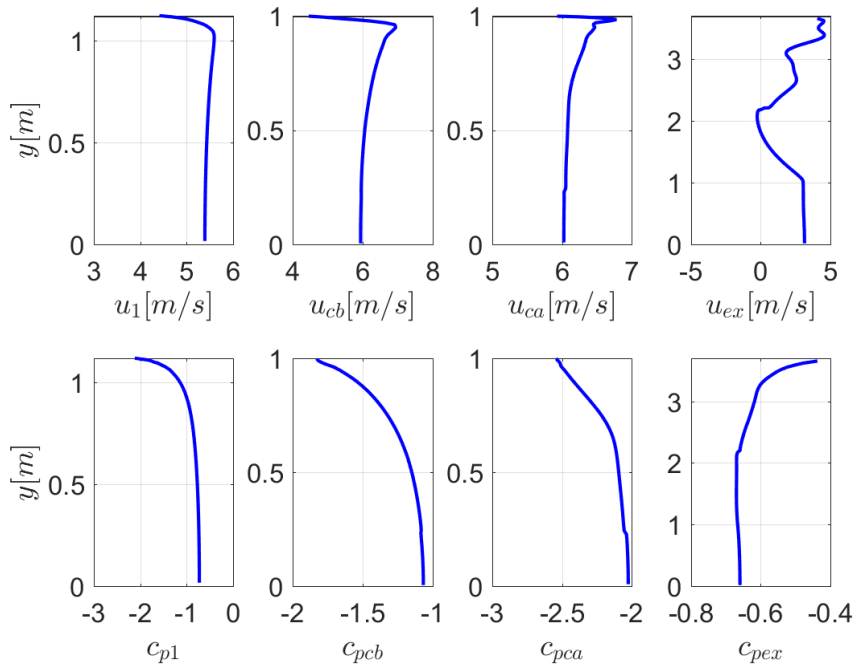
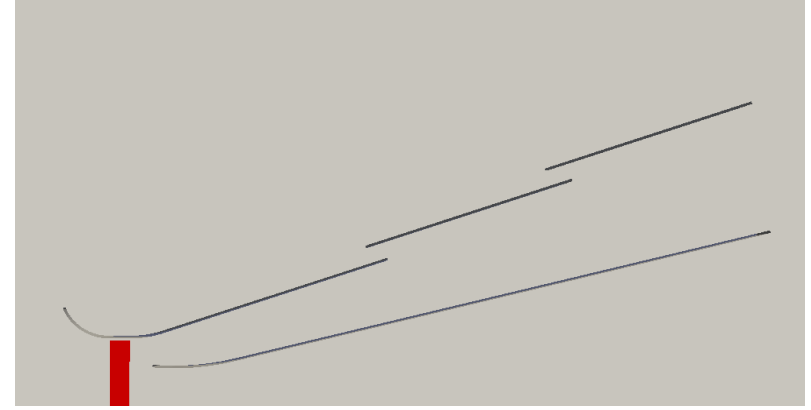
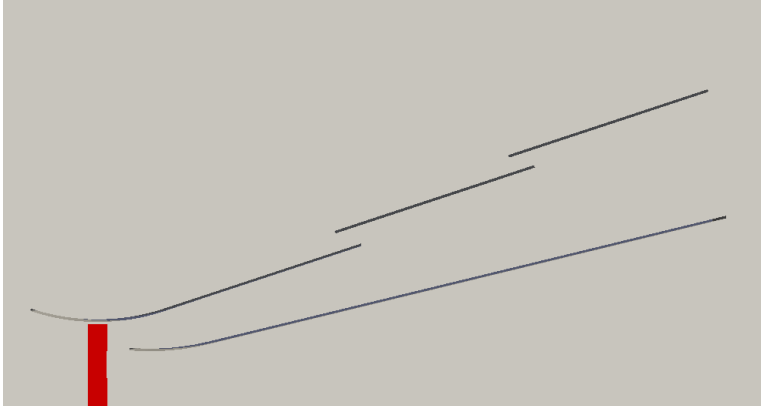
2a



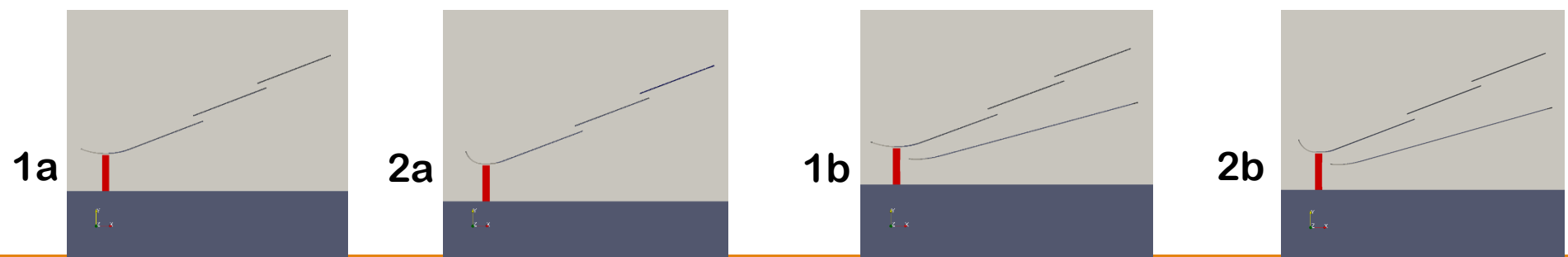
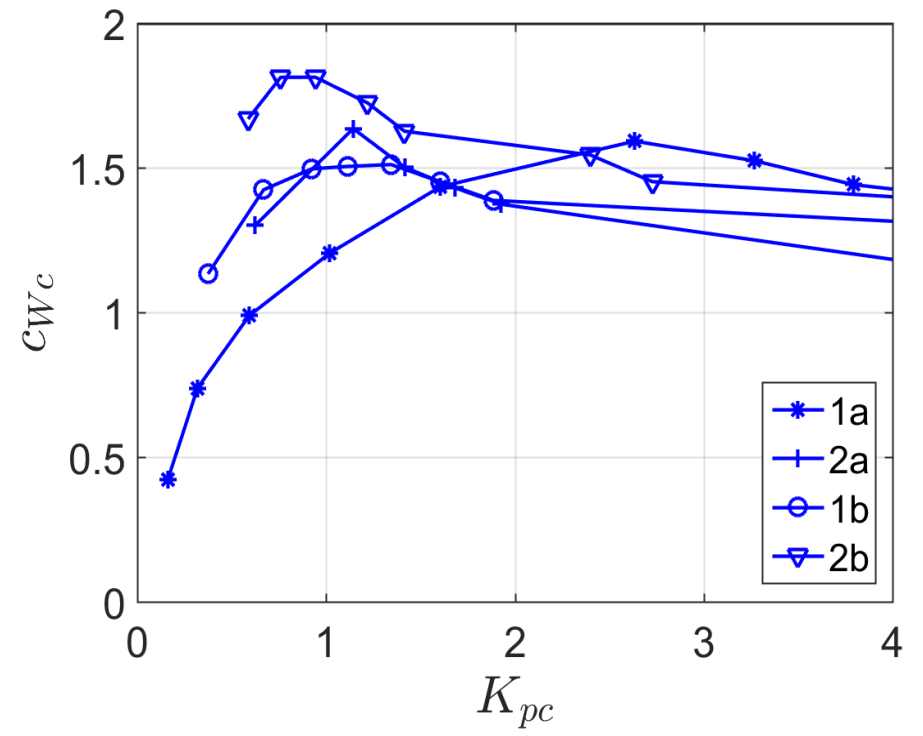
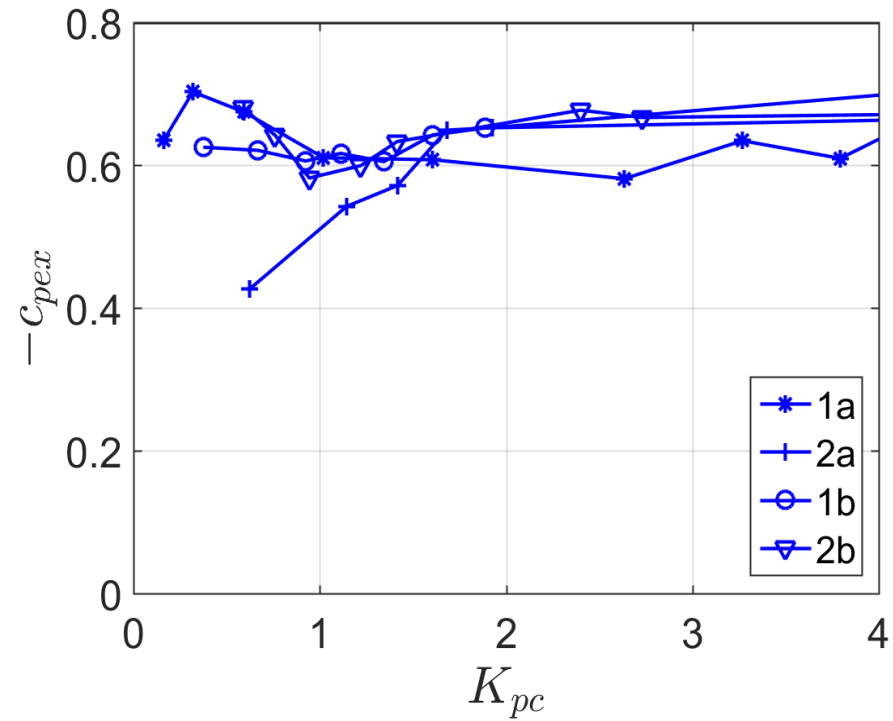


# Velocity and pressure coefficient profiles





# Pressure coefficient at the turbine exit and specific power extracted



# 5. Conclusions

- The simulations show the importance of a well design entrance to reduce pressure losses due to flow detachment.
- Comparing configurations 1a) and 1b), the effect of the vane is a reduction of the energy extracted. However, once the entrance was improved, the vane implies an increase of the energy extracted.
- The obtained values of  $c_{Wc}$  exceed the Betz limit for all configurations.
- When the efficiency of the entrance increase, the maximum of  $c_{Wc}$  occurs for smaller values of  $K_{pc}$ , which is in accordance to Lawn (2003).



# Thank you for your attention

